

#### HVAC Designer Responsibilities:

•	• Complete one National HVAC Design Report for each building which includes system design for all unique unit plans and common spaces. For
	projects with multiple buildings, one National HVAC Design Report per building or per project is permitted. <sup>1</sup>

- Obtain efficiency features (e.g., window performance, insulation levels, and infiltration rate) from the builder, architect, or Rater.<sup>2</sup>
- Provide the completed National HVAC Design Report to the Rater and the person / company completing the National HVAC Functional Testing Checklist.<sup>2</sup>

1. Design Overview							
1.1 Designer name:	Designer	company:			Date:		
1.2 Select which party you are providing these design servi	ces to: 🛛 Build	er / Develope	er 🗆 FT Agent	MEP / Crede	entialed HVAC c	ontrac	tor
1.3 Name of company you are providing these design servi	ces to (if differe	nt than Item ?	1.1):				
1.4 Building address:	City: _			_ State:	Zip code:		
2a. Dwelling Unit & Common Space Mechanical Ven						Desi	igner ified
Airflow:							
2.1 Dwelling unit ventilation airflow design rate & run-time r Prescriptive Path Only: Rates shall not exceed 2013 rate			tion 4 of ASHR	AE 62.2 <sup>7</sup> –		C	
2.2 Common space outdoor airflow design rate meet the re ERI and Prescriptive Path Only: Rates shall not exceed				10		Γ	
2.3 Access points to measure airflow rate and inspect outdo	oor air dampers	are provided	and accessible	e by the Rater. <sup>2,</sup>	11	[	
List unique unit plan for which 62.2 ventilation rates were calculated in the spaces to the right: <sup>12</sup>							
2.4 # of bedrooms:							
2.5 Square footage:							
2.6 Ventilation airflow rate required by ASHRAE 62.2:							
2.7 Ventilation airflow rate designed:							
2.7.1 If applicable, run-time per cycle (minutes):							
2.7.2 If applicable, cycle time (minutes):							
List common space for which 62.1 ventilation rates were calculated in the spaces to the right: <sup>11, 12, 13</sup>							
2.8 Ventilation airflow required by ASHRAE 62.1 (CFM): 10							
2.9 Ventilation airflow designed (CFM): 10							
System Type & Controls:							
List Ventilation System ID in the spaces to the right: <sup>12</sup>							
2.10 Specified system type: (e.g., supply, exhaust, balanced, ERV, HRV)							
2.11 Manufacturer:							
2.12 Model Number:							
2.13 # installed in the building:							
2.14 Spaces each fan serves (i.e., single, multiple)							
2.15 Area / space(s) that system serves: (e.g., Unit A kitchens, corridor, community room)							
2.16 Specified control location: (e.g., Master bath, utility):							
2.17 Specified controls allow the systems to operate autom specified and also labeled if its function is not obvious (e.g. the ventilation equipment). In townhouses only, this control units, the override control is not required to be readily accessible to other that the control be readily accessible to other the there are accessible to other the there are accessible to accessib	, a label is requi must be readily ssible to the occ	red for a togo accessible to cupant. Howe	gle wall switch, o the occupant. ver, in such cas	but not for a swi In all other mult ses, EPA recom	tch that's on ifamily dwelling mends but	[	
2.18 For any outdoor air inlet designed to connect to the dw using a motorized damper during ventilation off-cycle and c			ecified controls	automatically re	strict airflow		□ N/A
Sound:							
2.19 If located in the dwelling unit, the fan of the specified system is rated $\leq$ 3 sones if intermittent and $\leq$ 2 sones if continuous, or exempted. <sup>15</sup>							□ N/A
Efficiency:							
2.20 If dwelling-unit Vent System controller operates the dw the fan type in Item 4.12 is ECM / ICM, or the controls will r hours. <sup>16</sup>							□ N/A
2.21 If in-unit bathroom fans or in-line fans are specified as ENERGY STAR certified. <sup>17</sup>	part of the Dwe	elling Unit Me	chanical Ventila	ation System, the	en they are		□ N/A



						welling Unit Me ecified to meet									□ N/A
Air Inlet Loca	ations: (Co	omplete th	is section if	system has s	pecifie	d air inlet locati	on(s); other	wise	e check	"N/A".) <sup>19</sup>			-	Ver	igner ified N/A
2 23 Inlet(s) n	ull ventilat	tion air dire	actly from ou	tdoors and n	ot from	attic, crawlspa	co darado	ora	diacon	t dwelling u	init				
2.24 Inlet(s) a	re ≥ 2 ft. a	bove grad	le or roof de	ck; ≥ 10 ft. of	stretch	ed-string distance	nce from kno	own	contan	nination so		e.g., s	stack,		
2.25 Inlet(s) a			-					ing a		•				1	
						em(s) are desig	aned that me	echa	anically	exhaust ai	r from	each			
						entilation risers							ates. <sup>21</sup>		
Location			ous Rate			Intermittent	Rate 22					Exha	aust Far	і Тур	е
Kitchen	Airflow	25	, based on ki ive in Fn. 23)	tchen volume	2 <sup>23, 24,</sup>	≥ 100 CFM an ACH based or	nd, if not inte n kitchen vol	grat lume	ed with e <sup>23, 24, 2</sup>	range, als	o≥5	🗆 Int	ontinuous ermittent unit fan		
	Sound	Recomm	ended if in-u	nit: ≤ 1 sone		Recommende	d if in-unit: ≤	≤3 s	ones				entral / sh	nared	fan
Bathroom	Airflow	≥ 20 CFN	Л			≥ 50 CFM						□ Co	ntinuous	;	
	Sound	Required	l if in-unit: ≤ 2	2 sones		Recommende	d if in-unit: ≤	≤3 s	ones			🗆 In-	ermittent unit fan entral / sh		fan
2c. Common space						- System(s) are	e designed t	hat r	mechar	nically exha	aust air	from	each	[	
Location		ASHRA Rate	E 62.1	Design Rat	e	Location		,	ASHR	AE 62.1 Ra	ate	Desi	gn Rate	I	
Janitor Room		1 cfm/ft <sup>2</sup>				Common space	ce kitchen 27	5	50 cfm /	/ 100 cfm					
Trash / Recyc Room	ash / Recycling om 1 cfm/ft <sup>2</sup>			Common space	ce bathroom	<sup>28</sup> 5	50 cfm	per toilet /	urinal						
Parking Gara	ge		/ft <sup>2</sup> , standby /ft <sup>2</sup> , full-on			□ Shared gar	age exhaust	t fan	contro	ls include (	CO and	NO2	sensors		
3. Heating &	Cooling	Loads													
		-				ucted split AC,	-						-		N/A
	es only: L	oads mus	t be calculate	ed room-by-ro	oom.	□ 2013 / 2017								per A	√HJ <sup>30</sup>
						ds is unit-speci					re than	one u	unit: <sup>31</sup>		
	design (If	f the top flo	oor unit with	the greatest	CFAar	ups for this buind window area	a results in to	otal I	heat ga	in <18 kBt	uh, it m	nay rep	oresent a	all oth	er
			-			or two-speed and 75°F for c		beed	1 & <25	KBlun.					
	<b>y</b>					and <u>www.enei</u>	-	21/20	docian	tompe ) 31				L	
	-	-	ry selected: _	5. (See 1 000	1016 33				season			leatin	g season		°F
List the unit			-	culated: 12				inig	000001	·· ·			9 000001	<u> </u>	'
3.5 Location of														-	
3.6 Number of														+	
3.7 Total occu														1	
3.8 Conditione		, ,												1	
3.9 Window a	rea used i	n loads: 31	, 36											1	
3.10 Predomir	nant windo	ow SHGC	used in load	s: <sup>31, 37</sup>											
3.11 Infiltration	n (ACH / A	ACH50 / C	FM) used in	loads: 38											
3.12 Mechanie	cal ventila	tion (CFM	) used in loa	ds: <sup>31</sup>											
3.13 Non-occu and lighti		rnal gains in loads (B		equipment											
3.14 Door orie				/, NW): <sup>32</sup>										1	
3.15 Sensible	Heat Gair	n At Desig	n Conditions	(kBtuh): <sup>31</sup>	_							_ 1			
3.16 Latent He	eat Gain A	t Design (	Conditions (k	Btuh):											
3.17 Total Hea	at Gain at	Design Co	onditions (kB	tuh): <sup>31</sup>											
3.18 Total Hea	at Loss at	Design Co	onditions (kB	tuh):											



3.19 Common Space Heating & Coo	ling Loads <sup>12</sup> (rec	uired for all c	common s	pace heatir	ng and cool	ling systems)		Desig Veri	fied
Common Space Name:	Design Condition	ons: Total Hea	at Gain:	(kBtu	h) Tota	al Heat Loss: <sub>-</sub>	(kBtu	ıh)	
Common Space Name:	Design Condition	ons: Total Hea	at Gain:	(kBtu	h) Tota	al Heat Loss: _	(kBtu	ıh)	
Common Space Name:	Design Condition	ons: Total Hea	at Gain:	(kBtu	h) Tota	al Heat Loss: _	(kBtu	ıh)	
3.20 Building Heating & Cooling Loa	ads 12 (only require	ed when shar	ed syster	ns such as	central boil	ers or chillers	s are specifie	ed.) □	N/A
System Name:	Design Condition	ons: Total Hea	at Gain:	(kBtu	h) Tota	I Heat Loss:	(kBtu	h)	
System Name:	Design Condition	ons: Total Hea	at Gain:	(kBtu	h) Tota	I Heat Loss:	(kBtu	h)	
4. Heating & Cooling Equipment Sel	0			(	,			,	
4.1 Equipment selected per □ ACCA Ma		ot applicable.	Other:		. (See Fo	ootnote 39)			]
4.2 Prescriptive Path: Equipment serving Exhibit X of the National Rater Field Che	dwelling units, cor	nmon spaces	, and gara		e efficiency	levels specifie	ed in the		□ N/A
4.3 ERI Path: Equipment serving common the Exhibit X of the National Rater Field	n spaces and gara	ges but not se	erving dwe	lling units m	eet the effic	ciency levels s	pecified in		□ N/A
<b>Cooling Equipment</b> <sup>12</sup> (Complete all a (columns), identical data is not require									ces N/A
List Cooling Equipment ID in the spaces duplicating as needed for each uniq	to the right;								
4.4 Equipment type: (e.g., PTAC / AC, C WLHP / GSHP / ASHP / VRF)	hiller / CT, PTHP /								
4.5 Area / Space(s) that system serves:									
4.6 Chiller / condenser / outdoor unit ma	nufacturer:								
4.7 Chiller / condenser / outdoor unit mo	del #:								
4.8 Evaporator / indoor unit manufacture	r:								
4.9 Evaporator / indoor unit model #:									
4.10 AHRI reference #: 41									
4.11 Rated efficiency:									
4.12 Evaporator fan type: PSC, ECM / IC	M. Other								
4.13 Compressor speed: Single, Two, Va									
4.14 Turn down ratio (for variable speed									
4.15 Latent capacity at design conditions									
4.16 Sensible capacity at design condition	· /								
4.17 Total capacity at design conditions	· · ·								
4.18 Cooling sizing % = Total capacity (I	,								
by Total Heat Gain of space(s) in Item 4.	5: <sup>29</sup>								
4.19 Meets cooling sizing limit: (see belo N/A) <sup>29,31</sup>									
4.20 If "B", list Load sensible heat ratio = heat gain (Item 3.15) / Max. total heat ga									
4.21 If "B", calculate HDD / CDD ratio: <sup>43</sup>									
			1	essor Type	(Per Item	•			
Equipment Type & Climate Condition	Single-Sp	beed		Two-Speed		Va	ariable-Speed	t	
A: For Cooling-Only Equipment or For Cooling Mode of Heat Pump in Condition A Climate <sup>43</sup>	Recommended Allowed: 90		1	nmended: 90 owed: 90 – 7			nmended: 90 owed: 90 – 10		)
B: For Cooling Mode of Heat Pump in Condition B Climate <sup>43</sup>	90% - 100%, pl	us 15 kBtuh	90% -	100%, plus	15 kBtuh	90% - 100%, plus 15			1
C: For low-load spaces (≤15 kBtuh) <sup>44</sup>	≤ 20 kE	Btuh							
D: For low-load spaces (≤18 kBtuh) <sup>44</sup>				≤ 25 kBtuł	1		≤ 25 kBtuh		



Heating Equipment <sup>12</sup> (Complete all applicable items, no multiple spaces (columns), identical data is not required to							Designer Verified	
check "N/A".)	•			·	U		□ N/A	
List Heating Equipment ID in the spaces to the right; duplicating as needed for each unique space served:								
4.22 Electric equipment type: PTHP, WLHP, GSHP, ASHP, VRF, Boiler, Furnace, Electric Resistance								
4.23 Gas Equipment type: HW PTAC / fan coil, Gas-Fired PTAC, Boiler, Furnace								
4.24 Area / Space(s) that system serves:								
4.25 Manufacturer:								
4.26 Model Number:								
4.27 AHRI reference #: 41								
4.28 Rated efficiency:								
4.29 Equipment output capacity (kBtuh): 45								
4.30 Air-source heat pump output capacity (17°F) (kBtuh):								
4.31 Type of Venting: Natural Draft, Mechanically Drafted, Direct Vent <sup>46</sup>								
4.32 Furnace heating sizing % = Total capacity (Item 4.29) divided by Total Heat Loss of space(s) in Item 4.24: <sup>29</sup>								
4.33 Meets furnace sizing limit: (see below for A, B, C, or N/A) $^{\rm 29}$								
A: For low-load spaces	(≤ 10 kBtuh	), furnace c	utput capaci	ty is ≤ 40 kB	tuh			
B: When Used for Heating Only	B: When Used for Heating Only C: When Paired With Cooling							
100 – 400%		Reco	ommended: 1	00 – 140%	Allowed: 1	00 – 400%		
Equipment Controls								
4.34 All equipment controls below have been included where	e applicable	in the HVA	C Design.					
4.35 All heating and cooling systems serving a dwelling unit	shall have t	hermostatio	controls wit	nin the dwell	ing unit.			
4.35.1 Prescriptive Path: Dwelling unit thermostats are pro								
4.36 Stair and elevator shaft vents shall be equipped with m operation and are interlocked to open as required by fire and				f being auto	matically clos	ed during no	rmal building	
4.37 Freeze protection systems, such as heat tracing of pipi heaters shall include automatic controls capable of shutting Where heat tracing is specified for freeze protection, control required.	off the syste	ms when p	ipe wall or ga	arage / plenu	im temperatu	res are above	e 40°F.	
4.38 Snow- and ice-melting systems shall include automatic 50°F and no precipitation is falling, and an automatic or man the potential for snow or ice accumulation is negligible.								
Hydronic Distribution Requirements - Applies to heat	ing or cooliı	ng systems	serving mo	re than one	dwelling uni	t 🗆	I N/A	
4.39 All hydronic distribution requirements below have been	included wh	nere applica	ble in the H	AC Design.				
4.40 All terminal heating and cooling distribution equipment distribution pump, so that heated or cooled fluid is not delive thermostat.								
4.41 Terminal units must be equipped with pressure indeper	ndent baland	cing valves	or pressure i	ndependent	control valve	s.		
4.42 Piping of a heating or cooling system (e.g., steam, hot of ASHRAE 90.1-2007, Table 6.8.3. Construction documents in through planks or any other penetrations and shall specify the Heating System: Pipe size: inches Insulation thic Cooling System: Pipe size:	nust accoun nat the pipin kness: i	t for piping g must be ir nches	total thicknes	s including i	required insul	ation when p		
4.43 For circulating pumps serving hydronic heating or cooli exceed <u>efficiency standards for NEMA Premium</u> ™ motors.	ng systems	with three-p	hase motors	s, 1 horse-po	wer or larger	, motors shal	meet or	
4.44 If a variable speed pumping system is installed, system such as a minimum flow bypass valve or 3-way valves on sp			lead-heading	and a metl	nod of water f	low bypass is	s provided,	
4.45 For shared boilers, chillers, and cooling towers, temper clearly shown on the drawings. A complete sequence of ope condensing boilers, design return temperature is indicated a	erations for a	all systems i	ndicating rec	ommendatio	ons for all set	ooints is prov	ided. For	



5. Dwelling Unit Duct Design (Comp	lete if heating or cooling	equipr	ment will be installed with c	lucts; otherwis	e check "N/A".)	Designer Verified
						D N/A
5.1 Duct system designed for the equipn Townhouses only: Duct system must		•		ther:		
5.2 Room-by-room design airflows docu	mented below (which mus	t sum f	to the mode with the higher	Design HVAC fa	an airflow). 12, 47, 48	
Name of the unit plan:			e of the unit plan:	0	,	
Design HVAC fan airflow: 49			n HVAC fan airflow: 49			
Cooling mode CFM Heatin		CFM				
Design HVAC fan speed setting (e.g., low, medium, high): <sup>50</sup> Cooling mode Heating mode			n HVAC fan speed setting ( ng mode	e.g., low, mediu Heating mode		
Design total external static pressure (corresponding to the mode with the higher airflow above): <sup>51</sup> IWC			n total external static pressu he higher airflow above): <sup>51</sup>		ing to the mode	
Room Name	Design Airflow (CFM)		Room Name		Design Airflo	w (CFM)
1		1				
2		2				
3		3				
4		4				
5		5				
6		6				
7		7				
8		8				
9		9				
10		10				
Total for all rooms		<u> </u>		al for all rooms		
6. Duct Quality Installation – Applies					Unless Noted in	
6.1 Applicable duct quality installation re						
6.2 Ductwork specified without kinks, sha	arp bends, compressions,	or exc	essive coiled flexible ductwo	ork. <sup>52</sup>		
6.3 All supply and return ducts not in cor	nditioned space, including	conne	ctions to trunk ducts, are ins	ulated to ≥ R-6	53	
6.3.1 Prescriptive Path: Dwelling unit Design.	ductwork meets the locati	ion and	l insulation requirements spe	ecified in the EN	NERGY STAR MF	Reference
Dwelling Unit						
<ol> <li>MERV 6+ filter(s) specified for each d supplied outdoor air passes through filte or building maintenance staff. Filter acce</li> </ol>	r(s) prior to conditioning, a	and loc	ated to facilitate access & re	gular service b		
6.5 Ductwork air-sealing specified such t ft <sup>2</sup> at final, or if there are no ducted return Townhouses only, Rater-measured duct	ns, ≤ 3 CFM25 per 100 ft²	of CFA	A at rough-in or ≤ 6 CFM25 p	per 100 ft <sup>2</sup> at fin	ough-in or ≤ 8 CFI al. <sup>54</sup> Additionally,	M25 per 100 for
6.6 All bedrooms provided with transfer ( 150 CFM (as reported in Item 5.2) are sp body of the dwelling unit when all air har specified to achieve a Rater-measured p	pecified to achieve a Rater ndlers are operating. Town	r-meas house	ured pressure differential ≥ s only: In addition, bedroom	- 5 Pa and ≤ 5 I	Pa with respect to	the main
Common Space and Central Exhaust						
6.7 Duct design specifies that all supply, joints, longitudinal seams, and duct wall		vork ar	nd all plenums serving comm	non spaces sha	II be sealed at all t	ransverse
6.8 Central exhaust systems (that serve exceed 25% of exhaust fan flow at rough ductwork between the fan and the grilles	n-in (e.g., including trunks,					



#### Footnotes:

- 1. This report shall represent system design for all unique unit plans, common spaces, and where applicable, parking garages. The term 'common space' refers to any spaces in the building being certified that serve a function in support of the residential part of the building that is not part of a dwelling or sleeping unit. This includes spaces used by residents, such as corridors, stairs, lobbies, laundry rooms, exercise rooms, residential recreation rooms, and dining halls, as well as offices and other spaces used by building management, administration or maintenance in support of the residents. As an alternative, for dwelling units, designers may instead choose to complete a Single-Family New Homes National HVAC Design Report for each unique unit plan, if room-by-room loads are calculated using Unabridged ACCA Manual J v8. Sections 4 and 5 must be completed in either Design Report unless exempted by this Report. All other systems, including all systems serving common spaces, must be documented in this Design Report. This report is designed to meet ASHRAE 62.1-2010 or later, ASHRAE 62.2-2010 or later, and ANSI / ACCA's 5 QI-2015 protocol, thereby improving the performance of HVAC equipment in new multifamily buildings when compared to multifamily buildings built to minimum code. However, these features alone cannot prevent all ventilation, indoor air quality, and HVAC problems (e.g., those caused by a lack of maintenance or occupant behavior). Therefore, system designs documented through the use of this report are not a guarantee of proper ventilation, indoor air quality, or HVAC performance.
- The term 'Rater' refers to the person(s) completing the third-party verification required for certification. The person(s) shall: a) be a Certified Rater, Approved Inspector, as defined by ANSI / RESNET / IECC 301, or an equivalent designation as determined by a Home Certification Organization (HCO) or Multifamily Review Organization (MRO); and, b) have attended and successfully completed an EPA-recognized training class. See <u>www.energystar.gov/mftraining</u>.
- 3. As defined by ANSI / RESNET / ICC 301-2019, a Dwelling Unit Mechanical Ventilation System is a ventilation system consisting of powered ventilation equipment such as motor-driven fans and blowers and related mechanical components such as ducts, inlets, dampers, filters and associated control devices that provides dwelling-unit ventilation at a known or measured airflow rate.
- 4. The dwelling-unit mechanical ventilation system shall have at least one supply or exhaust fan with associated ducts and controls. Local exhaust fans are allowed to be part of a dwelling-unit mechanical ventilation system. Designers may provide supplemental documentation as needed to document the system design. For example, for Item 2.7, designers are permitted to provide multiple combinations of a design ventilation airflow rate, run-time per cycle, and cycle time. When multiple combinations are provided, the Rater will be required to first assess the run-time setting of the installed system and use that to determine the corresponding design ventilation rate. The Rater-measured ventilation rate then must fall within the program-specified tolerance relative to that design ventilation rate.
- 5. In "Warm-Humid" climates as defined by 2009 IECC Figure 301.1 (i.e., CZ 1 and portions of CZ 2 and 3A below the white line), it is recommended, but not required, that equipment be specified with sufficient latent capacity to maintain indoor relative humidity at ≤ 60%.
- 6. Item 2.18 applies to any outdoor air inlet connected to the dwelling unit HVAC system, regardless of its intended purpose (e.g., for ventilation air, make-up air, combustion air). For example, if an outdoor air inlet connected to a ducted return is used as a dedicated source of outdoor air for an exhaust ventilation system (e.g., bath fan), the outdoor airflow must be automatically restricted when the exhaust fan is not running and in the event of an override of the exhaust ventilation system.

In dwelling / sleeping units in multifamily buildings, but not townhouses, automatic restriction of airflow is exempted if a manual shutoff damper is used with a continuous exhaust ventilation system and is readily-accessible, labeled as the override, and not used as a balancing damper.

Note that a Rater will generally measure the ventilation rate at the highest HVAC fan speed applicable to ventilation mode (e.g., if the inlet only opens when the HVAC is in 'fan-only' mode, then test in this mode) to verify that it is  $\leq$  15 CFM or 15% above design value. If the inlet has a motorized damper that only opens when the local mechanical kitchen exhaust is turned on, then testing is not required. As an alternative, measurement of the outdoor airflow can be waived if a Constant Airflow Regulating (CAR) damper with a manufacturer-specified maximum flow rate no higher than 15 CFM or 15% above the ventilation design value is installed on the inlet.

- 7. Airflow design rates and run-times shall be determined using ASHRAE 62.2-2010 or later. Designers are permitted, but not required, to use published addenda and/or more recent editions of the standard to assess compliance. The year of the standard that is used shall be listed in the space provided. For dwelling units, the minimum ventilation rate required by ASHRAE 62.2 can be calculated using either Equation 4.1a or Table 4.1a. For sleeping units, the following equation must be used to determine minimum airflow rates: 0.01 x Conditioned Floor Area + 7.5 x (number of beds).
- 8. Where the Exhaust Fan Type in Section 2b indicates "Continuous" for both Bathroom and Kitchen, the Rater may use this equation to determine the maximum ventilation rate allowed: 30 CFM x number of bathrooms + 75 CFM.
- 9. Airflow design rates shall be determined using ASHRAE 62.1-2010 or later. Designers are permitted, but not required, to use published addenda and/or more recent editions of the standard to assess compliance. The year of the standard that is used shall be listed in the space provided.
- 10. The following spaces require outdoor air ventilation: corridors, offices, break rooms, gyms, fitness centers, exercise rooms, lobbies, community rooms, meeting rooms, multi-purpose rooms, lounges, laundry rooms, swimming pools, daycares, classrooms, shared or commercial kitchens, shared dining rooms, and computer rooms.
- 11. For permits on or before 01/01/2024, where outdoor air is supplied to a common space via a PTAC or PTHP, in lieu of measurement, the design CFM shall meet or exceed the ventilation rates required by ASHRAE 62.1-2010 and the space served by the PTAC or PTHP shall have at least one operable window. For permits after 01/01/2024, both the runtime and measurement of outdoor air through these systems will be required to demonstrate compliance with ASHRAE 62.1-2010 or alternative ventilation system specified (e.g., ducted supply).
- 12. If the tables provided cannot accommodate all the unit plans, spaces, or systems in the building, use the tables in Appendix A to supplement the Design Report.
- 13. List each individual common space separate from other spaces, such that when reporting airflow for Items 2.8 and 2.9, compliance for each space can be demonstrated. For example, list an office space separate from a community room, even if these spaces are served by the same system and even if the outdoor air rates required are the same. Similarly, where a space is repeated in the building, such as a corridor, report each space by floor (e.g., FL1 Corridor, FL2 Corridor). Rather than list these values in this report, as an alternative, the HVAC Designer is permitted to submit the values in a separate document or file. Where the building has total corridor space ≤ 250 ft<sup>2</sup> and does not contain any of the other common spaces which require outdoor air per Item 2.2, outdoor air is not required to be provided to the corridor and "N/A" may be entered for Item 2.9.
- 14. In addition, consult manufacturer requirements to ensure return air temperature requirements are met.



- 15. Dwelling-unit mechanical ventilation fans shall be rated for sound at no less than the airflow rate in Item 2.7. Fans exempted from this requirement include HVAC air handler fans, remote-mounted fans, and intermittent fans rated ≥ 400 CFM. To be considered for this exemption, a remote-mounted fan must be mounted outside the habitable spaces, bathrooms, toilets, and hallways and there shall be ≥ 4 ft. ductwork between the fan and intake grill. Per ASHRAE 62.2-2010, habitable spaces are intended for continual human occupancy; such space generally includes areas used for living, sleeping, dining, and cooking but does not generally include bathrooms, toilets, hallways, storage areas, closets, or utility rooms.
- 16. Note that the 'fan-on' setting of a thermostat would not be an acceptable controller because it would continuously operate the HVAC fan.
- 17. Bathroom fans with a rated flow rate ≥ 500 CFM and heat/energy recovery ventilation fans are exempted from the requirement to be ENERGY STAR certified.
- 18. As an alternative to meeting or exceeding the efficiency standards for NEMA Premium motors, documentation that an exhaust fan motor has a fan energy index (FEI) ≥ 1.2 at the design point of operation OR a fan efficacy ≥ 1.1 CFM/Watt is permitted.
- 19. Without proper maintenance, ventilation air inlet screens often become filled with debris. Therefore, EPA recommends, but does not require, that these ventilation air inlets be located so as to facilitate access and regular service by the building maintenance staff.
- 20. Two alternatives to the required 10 ft. distance are provided: 1) inlets providing outdoor air to a dwelling unit are permitted to be ≥ 5 ft. of stretched-string distance from outlets of both exhaust dwelling-unit mechanical ventilation systems and local mechanical exhaust systems, and 2) the outlet and inlet of ERV's and HRV's may use a smaller distance if allowed by the manufacturer of the system. If the second alternative is used, the manufacturer's instructions shall be collected for documentation purposes.
- 21. Continuous bathroom local mechanical exhaust fans shall be rated for sound at no less than the design airflow rate. Intermittent bathroom and both intermittent and continuous kitchen local mechanical exhaust fans are recommended, but not required, to be rated for sound at no less than the design airflow rate. Per ASHRAE 62.2-2010, an exhaust system is one or more fans that remove air from the building, causing outdoor air to enter by ventilation inlets or normal leakage paths through the building envelope (e.g., bath exhaust fans, range hoods, clothes dryers). Per ASHRAE 62.2-2010, a bathroom is any room containing a bathtub, shower, spa, or similar source of moisture.
- 22. An intermittent mechanical exhaust system, where provided, shall be designed to operate as needed by the occupant. Control devices shall not impede occupant control in intermittent systems.
- 23. Where 5 ACH is selected, kitchen volume shall be determined by drawing the smallest possible rectangle on the floor plan that encompasses all cabinets, pantries, islands, peninsulas, ranges / ovens, and the kitchen exhaust fan, and multiplying by the average ceiling height for this area. In addition, the continuous kitchen exhaust rate shall be ≥ 25 CFM, per 2009 IRC Table M1507.3, regardless of the rate calculated using the kitchen volume. Cabinet volume shall be included in the kitchen volume. As an alternative to 5 ACH for Dwelling Units and Sleeping Units (but not Townhouses), 50 CFM of continuous exhaust is permitted to be used, regardless of kitchen volume. In such cases, the edge of the exhaust fan or intake grille shall be located within 10 ft of the edge of the range, as measured horizontally on the floor plan.
- 24. While not required, the prescriptive duct sizing requirements in Table 5.3 of ASHRAE 62.2-2010 are recommended to be used for kitchen exhaust fans based upon the rated airflow of the fan at 0.25 IWC.
- 25. As an alternative, dwelling units are permitted to use a continuous kitchen exhaust rate of 25 CFM per 2009 IRC Table M1507.3, if they are either a) PHIUS+ or PHI certified, or b) provide both dwelling unit ventilation and local mechanical kitchen exhaust using a balanced system, and have a Rater-verified whole-building infiltration rate ≤ 1.0 ACH50 or ≤ 0.05 CFM50 per ft<sup>2</sup> of Enclosure Area. 'Enclosure Area' is defined as the area of the surfaces that bound the volume being pressurized / depressurized during the test.
- 26. All intermittent kitchen exhaust fans must be capable of exhausting at least 100 CFM. In addition, if the fan is not part of a vented range hood or appliance-range hood combination (i.e., if the fan is not integrated with the range), then it must also be capable of exhausting ≥ 5 ACH, based on the kitchen volume.
- 27. For continuous system operation, the lower rate may be used. Otherwise, use the higher rate. Commercial kitchens shall be designed to provide a minimum continuous rate of 0.70 cfm/ft<sup>2</sup>.
- 28. As an alternative, for a toilet room intended to be occupied by one person at a time, a minimum continuous rate of 25 cfm is permitted.
- 29. This section / item applies to split air conditioners, unitary air conditioners, air-source heat pumps, and water-source (i.e., geothermal) heat pumps up to 65 kBtuh with forced-air distribution systems and to furnaces up to 225 kBtuh with forced-air distribution system serving individual dwelling units. Forced-air distribution systems are those that supply air through ductwork exceeding 0 ft. in length. For VRF air conditioners or heat pumps, the capacity of the system is the rated cooling capacity of the outdoor unit. This section / item is recommended, but not required for non-ducted systems, such as non-ducted mini-splits, multi-splits, PTHP's, or PTAC's.
- 30. Select "2013 / 2017 ASHRAE Fundamentals" if using Chapter 17 of the 2013 or 2017 ASHRAE Handbook of Fundamentals. Select "Other per AHJ" if the Authority Having Jurisdiction where the unit will be certified mandates the use of a load calculation methodology other than Unabridged ACCA Manual J v8 or 2013 or 2017 ASHRAE Handbook of Fundamentals.
- 31. Check the box for "unit-specific design" if the design was created for the specific plan configuration (i.e., elevation, option, orientation, and county) of the unit to be certified. Check the box for "group design" if designs were created for unit plans that are repeated throughout the building with potentially different configurations (i.e., different elevations and/or orientations). Check the box for "worst-case design" if loads for the unit with the largest heat gain in the building are less than 18 kBtuh and are being used to represent all other units. Only one box may be checked. Regardless of the box checked, the system design as documented on this HVAC Design Report must fall within the following tolerances for the unit to be certified:
  - Item 3.4: The outdoor design temperature used in loads are within the limits defined at <u>www.energystar.gov/hvacdesigntemps</u>.
  - Item 3.6: The number of occupants used in loads is within  $\pm 2$  of the dwelling unit to be certified.
  - Item 3.7: Total occupant gains used in loads shall not exceed 645 Btuh per occupant.
  - Item 3.8: The conditioned floor area used in loads is between 100 ft<sup>2</sup> smaller and 300 ft<sup>2</sup> larger than the dwelling unit to be certified.
  - Item 3.9: The window area used in loads is between 15 ft<sup>2</sup> smaller and 60 ft<sup>2</sup> larger than the dwelling unit to be certified, or for dwelling units with > 500 ft<sup>2</sup> of window area, between 3% smaller and 12% larger.
  - Item 3.10: The predominant window SHGC is within 0.1 of the predominant value in the dwelling unit to be certified.



- Item 3.12: The mechanical ventilation rate used in loads is the same as the value in Section 2a for the given unit plan.
- Item 3.13: The sum of the internal gains associated with lighting and appliances used in loads shall not exceed 3,600 Btuh.
- Items 3.15 & 3.17: The sensible & total heat gain are documented for the configuration of the dwelling unit to be certified.
- Item 4.19: The cooling sizing % is within the cooling sizing limit selected.

Provide the National HVAC Design Report to the party you are providing these design services to (i.e., a builder / developer, Functional Testing Agent (FT Agent), and/or MEP / credentialed HVAC contractor) and to the Rater. The report is only required to be provided once per project / building. As long as a report has been provided that falls within these tolerances for the units to be certified, no additional work is required. However, if no report falls within these tolerances or if any aspect of the system design changes, then an additional report will need to be generated prior to certification. Buildings certified under Rev. 04 of the program requirements are permitted to use any Revision of the MFNC National HVAC Design Report.

Visit www.energystar.gov/hvacdesigntools for a tool to assist with group designs and for more information.

- 32. For each unique unit floorplan, document the loads for the configuration (e.g., level, orientation) that the dwelling unit might be built in. For example, if a unit plan will only be built in a specific level and orientation (e.g., top-floor, facing South), then the designer only needs to document the loads for this one configuration. To determine whether a unit floorplan is "unique", the guidance in ANSI 301-2019, Section 5.1.4.4.1 may be followed. Orientation represents the direction that the front door of the dwelling unit is facing. In Section 4, to calculate Cooling sizing % for each configuration of each unique floorplan, the same system may need to be duplicated in multiple columns.
- 33. Visit <u>www.energystar.gov/hvacdesigntemps</u> for the maximum cooling season design temperature and minimum heating season design temperature permitted for ENERGY STAR. For "County & State, or US Territory, selected", select the County and State or US Territory (i.e., Guam, Northern Mariana Islands, Puerto Rico, or US Virgin Islands), where the unit is to be certified. The same design report is permitted to be used in other counties, as long as the design temperature limits in those other counties meet or exceed the cooling and heating season temperature limits for the county selected. For example, if Fauquier County, VA, is used for the load calculations, with a 1% cooling temperature limit of 93°F, then the same report could be used in Fairfax County (which has a higher limit of 94°F) but not in Arlington County (which has a lower limit of 92°F). If a jurisdiction-specified design temperature is used that exceeds the limit in the ENERGY STAR Single-Family New Homes Design Temperature Limit Reference Guide, designers must submit a Design Temperature Exception Request. Visit www.energystar.gov/hvacdesigntemps for a copy of this form.
- 34. To determine the number of occupants among all HVAC systems in the dwelling unit, calculate the number of bedrooms, as defined below, and add one. This number of occupants must be within ± 2 of the dwelling unit to be certified.

A bedroom is defined by ANSI / RESNET / ICC 301-2014 as a room or space 70 ft<sup>2</sup> or greater size, with egress window and closet, used or intended to be used for sleeping. A "den", "library", or "home office" with a closet, egress window, and 70 ft<sup>2</sup> or greater size or other similar rooms shall count as a bedroom, but living rooms and foyers shall not.

An egress window, as defined in 2009 IRC section R310, shall refer to any operable window that provides for a means of escape and access for rescue in the event of an emergency. The egress window definition has been summarized for convenience. The egress window shall:

- have a sill height of not more than 44 inches above the floor; AND
- have a minimum net clear opening of 5.7 ft<sup>2</sup>; AND
- have a minimum net clear opening height of 24 in.; AND
- have a minimum net clear opening width of 20 in.; AND
- be operational from the inside of the room without the use of keys, tools or special knowledge.
- 35. The difference between the Conditioned Floor Area (CFA) used in the design and the actual dwelling unit to be certified must fall within the tolerance specified in Footnote 22, as verified by a Rater. Be advised, the Rater will calculate CFA using the definition in ANSI / RESNET / ICC 301-2019, which defines this value, in part, as the floor area of the Conditioned Space Volume within a building or Dwelling Unit, not including attics, and floors. the floor area of crawlspaces, and basements below air sealed insulated See https://codes.iccsafe.org/content/RESNET3012019P1/3-definitions- for the complete definition.
- 36. The difference between the window area used in the design and the actual dwelling unit to be certified must fall within the tolerance specified in Footnote 22, as verified by a Rater. Be advised, the Rater will calculate window area using the on-site inspection protocol provided in Normative Appendix B of ANSI / RESNET / ICC 301-2019, which instructs the Rater to measure the width and height of the rough opening for the window and round to the nearest inch, and then to use these measurements to calculate window area, rounding to the nearest tenth of a square foot. See <a href="https://codes.iccsafe.org/content/RESNET3012019P1/normative-appendix-b-inspection-procedures-for-minimum-rated-features">https://codes.iccsafe.org/content/RESNET3012019P1/normative-appendix-b-inspection-procedures-for-minimum-rated-features</a> for the complete protocol.
- 37. "Predominant" is defined as the SHGC value used in the greatest amount of window area in the dwelling unit.
- 38. Infiltration rate shall use "Tight" values for the cooling season infiltration rate and "Tight" values for the heating season infiltration rate, as defined by Table 5A or 5B of ACCA Manual J, Eighth Edition, Version Two. Alternatively, infiltration rate shall not exceed 0.24 air changes per hour.
- 39. Equipment shall be selected using the maximum total heat gain and the total heat loss in Section 3 per ACCA Manual S, Second Edition, except that cooling ranges above ACCA Manual S limits are temporarily allowed, per Item 4.19, and heating ranges above ACCA Manual S limits are allowed where heating and hot water are provided by the same equipment or where standby equipment is needed for redundancy, but only operate when the primary equipment is not operating. For equipment outside the scope of ACCA Manual S, "Other" may be indicated and the equipment sizing approach listed in the space provided.
- 40. Electric resistance limitations do not apply to heat pumps with integral supplemental or emergency electric resistance heating. EPA recommends but does not require that heat pumps have controls to limit the use of emergency or supplemental heat to heat pump failures or when the heat pump cannot meet the heating load. EPA also recommends but does not require that heat pumps in CZ 5-8 are ENERGY STAR certified cold-climate heat pumps. Electric resistance limitations do not apply to systems dedicated to heating outdoor air supplied by a mechanical ventilation system, as long as the space served is primarily heated by a non-electric-resistance system that meets the efficiency requirements noted in Exhibit X. Electric resistance limitations apply to garages, but do not apply to heated plenums meeting Item 4.37, or stairwells where automatic thermostatic controls prevent operation above 50°F.

- 41. If the equipment contains multiple components, the AHRI Reference # shall represent the rated efficiency of the specific combination of indoor and outdoor components. EPA recommends, but does not require, that the rating also encompass the furnace when such a rating is available. If an AHRI Reference # is not available, OEM-provided documentation shall be attached with the rated efficiency. For residential split air conditioners and heat pumps, the rated efficiency shall be for the specific combination of indoor and outdoor components of the air conditioner or heat pump, along with confirmation that the two components are designed to be used together. If the AHRI Reference # is reported in Item 4.10 (e.g., heat pumps), the AHRI Reference # does not need to be listed again in Item 4.27.
- 42. The full system capacity at design conditions, from OEM expanded performance data, shall be listed and shall include the capacity of all systems providing space cooling to the dwelling unit. For two-speed or variable-speed equipment, the full system capacity shall reflect the capacity at the maximum available compressor speed or when the compressor operates at the AHRI rating test speed, respectively.
- 43. Per ACCA Manual S, Second Edition, if the load sensible heat ratio is ≥ 95% and the HDD / CDD ratio is ≥ 2.0, then the Climate is Condition B, otherwise it is Condition A.
- 44. As an alternative for low-load dwelling units, a system match-up including a single-speed compressor with a total capacity ≤ 20 kBtuh is permitted to be used in spaces with a total cooling load ≤ 15 kBtuh. A system match-up including a two-speed or variable-speed compressor with a total capacity ≤ 25 kBtuh is permitted to be used in spaces with a total cooling load ≤ 18 kBtuh.
- 45. The full system capacity shall be listed for the heating system. For two-stage and modulating furnaces, the full system capacity shall reflect the maximum output available. For shared boilers, the full system capacity may exclude standby equipment needed for redundancy.
- 46. Per the 2009 International Mechanical Code, a direct-vent furnace or boiler is one that is constructed and installed so that all air for combustion is derived from the outdoor atmosphere and all flue gases are discharged to the outside atmosphere; a mechanical draft system is a venting system designed to remove flue or vent gases by mechanical means consisting of an induced draft portion under non-positive static pressure or a forced draft portion under positive static pressure; and a natural draft system is a venting system designed to remove flue or vent gases under non-positive static vent pressure entirely by natural draft. Naturally drafted equipment is only allowed if located in a space outside the pressure boundary, where the envelope assemblies separating it from conditioned space are insulated and air-sealed. For mechanically drafted boilers, make-up air sources must be mechanically closed when the boiler is not in operation.
- 47. Designers may provide supplemental documentation with room-by-room and total design airflows in lieu of completing Item 5.2. Sample supplemental documentation can be found at <u>www.energystar.gov/hvacdesigntools</u>.
- 48. Orientation-specific room-by-room design airflows are recommended, but not required, to distribute airflow proportional to load, thereby improving comfort and efficiency. While air-balancing of supply registers and return grilles is not required to be completed as part of HVAC Functional Testing, it is recommended that ducted HVAC systems be designed such that they can be balanced in the field (i.e., provide proper access to any and all balancing dampers, provide ducting and grille layouts such that accurate air measurements can be taken).
- 49. Design HVAC fan airflow is the design airflow for the blower in CFM, as determined using the manufacturer's expanded performance data. The Functional Testing Agent is required to measure the HVAC fan airflow using the mode with the higher airflow, within ± 15% of design.
- 50. Design HVAC fan speed setting is the fan speed setting on the control board (e.g., low, medium, high) that corresponds with the Design HVAC fan airflow.
- 51. Design total external static pressure is the pressure corresponding to the Design HVAC fan airflow, inclusive of external components (e.g., evaporator coil, whole-house humidifier, or ≥ MERV 6 filter).
- 52. Kinks are to be avoided and are caused when ducts are bent across sharp corners such as framing members. Sharp bends are to be avoided and occur when the radius of the turn in the duct is less than one duct diameter. Compression is to be avoided and occurs when flexible ducts in unconditioned space are installed in cavities smaller than the outer duct diameter and ducts in conditioned space are installed in cavities smaller than the outer duct diameter needed for acoustical control.
- 53. Item 6.3 does not apply to ducts that are a part of local mechanical exhaust or exhaust-only dwelling-unit ventilation systems. EPA recommends, but does not require, that all metal ductwork not encompassed by Section 6 (e.g., exhaust ducts, duct boots, ducts in conditioned space) also be insulated and that insulation be sealed to duct boots to prevent condensation.
- 54. Item 6.5 generally applies to the ducts of space heating, space cooling, and Dwelling Unit Mechanical Ventilation Systems. However, visual inspection is permitted in lieu of testing for the following system types: 1) a Dwelling Unit Mechanical Ventilation System not connected to the space heating or space cooling system, regardless of the number of dwelling units it serves; 2) a space heating or space cooling system for which the ducts and air handler are in conditioned space and the total supply duct length of the system, including all supply trunks and branches, is ≤ 10 ft; and 3) a space heating or space cooling system that serves more than one dwelling unit. In such cases, a Rater shall visually verify that all seams and connections are sealed with mastic or metal tape and all duct boots are sealed to floor, wall, or ceiling using caulk, foam, or mastic tape.
- 55. Duct leakage shall be determined and documented by a Rater in accordance with ANSI / RESNET / ICC 380. Leakage limits shall be assessed on a per-system, rather than per-dwelling unit, basis. For a duct system with one or two returns, the total Rater-measured duct leakage is permitted to be the greater of ≤ 4 CFM25 per 100 ft<sup>2</sup> of CFA or ≤ 40 CFM25 at 'rough-in' or the greater of ≤ 8 CFM25 per 100 ft<sup>2</sup> of CFA or ≤ 8 CFM25 at 'final'. For a duct system with three or more returns, the total Rater-measured duct leakage is permitted to be the greater of ≤ 6 CFM25 per 100 ft<sup>2</sup> of CFA or ≤ 60 CFM25 at 'rough-in' or the greater of ≤ 12 CFM25 per 100 ft<sup>2</sup> of CFA or ≤ 120 CFM25 at 'final'. For a duct system without any ducted returns, the total Rater-measured duct leakage is permitted to be the greater of ≤ 3 CFM25 per 100 ft<sup>2</sup> of CFA or ≤ 30 CFM25 at 'rough-in' or the greater of ≤ 6 CFM25 per 100 ft<sup>2</sup> of CFA or ≤ 60 CFM25 at 'final' and, the Rater-measured pressure difference between the space containing the air handler and the conditioned space, with the air handler running at high speed, is ≤ 5 Pa. For systems > 1 ton, increase by 1 Pa per half ton.
- 56. For the purpose of computing leakage allowance, at rough-in, the 'exhaust fan flow' shall be the lesser of the rated fan flow (i.e., nameplate rating) and 133% of the sum of the design exhaust airflow of the dwelling units served by that fan. At final, the 'exhaust fan flow' shall be the lesser of the rated fan flow (i.e., nameplate rating) and 143% of the sum of the design exhaust airflow of the dwelling units served by that fan. To calculate central exhaust duct leakage allowance, EPA recommends using worksheet 3b of the Multifamily Workbook. Measured fan flow (either at the fan itself or the total airflow measured from all exhaust grilles served by the fan) may be used in lieu of the rated fan flow to determine the leakage allowance. This test is not required of central exhaust systems serving clothes dryers but is required for the central exhaust portion of balanced systems such as HRVs and ERVs.



### Appendix A – Supplementary tables for Section 2 and 3

2a. Dwelling Unit & Common Space Mechanical Ver	ntilation Design 4, 5			
List unique unit plan for which 62.2 ventilation rates were calculated in the spaces to the right:				
2.4 # of bedrooms:				
2.5 Square footage:				
2.6 Ventilation airflow rate required by ASHRAE 62.2:				
2.7 Ventilation airflow rate designed:				
2.7.1 If applicable, run-time per cycle (minutes):				
2.7.2 If applicable, cycle time (minutes):				
		•	•	•
List common space for which 62.1 ventilation rates were calculated in the spaces to the right:				
2.8 Ventilation airflow rate required by ASHRAE 62.1:				
2.9 Ventilation airflow rate designed:				
				•
System Type & Controls:				
List Ventilation System ID in the spaces to the right:				
2.10 Specified system type: (e.g., supply, exhaust, balanced, ERV, HRV)				
2.11 Manufacturer:				
2.12 Model Number:				
2.13 # installed in the building:				
2.14 Spaces each fan serves (i.e., single, multiple)				
2.15 Area / space(s) that system serves: (e.g., Unit A kitchens, corridor, community room)				
2.16 Specified control location: (e.g., Master bath, utility):				
3. Heating & Cooling Loads				

3. Heating & Cooling Loads								
Dwelling Unit Heating & Cooling Loads (only required for ducted split AC, unitary AC, ASHP, WSHP, GSHP, and furnaces) <sup>28</sup> D N/A								
List the unit plan for which Loads were calculated:								
3.5 Location of Unit: top, mid, bottom, corner, interior								
3.6 Number of occupants used in loads: <sup>31, 34</sup>								
3.7 Total occupant gains (Btuh): 31								
3.8 Conditioned floor area used in loads: <sup>31, 35</sup>								
3.9 Window area used in loads: <sup>31, 36</sup>								
3.10 Predominant window SHGC used in loads: <sup>31, 37</sup>								
3.11 Infiltration (ACH / ACH50) used in loads: <sup>38</sup>								
3.12 Mechanical ventilation (CFM) used in loads:								
3.13 Non-occupant Internal gains (appliance, equipment and lighting) used in loads (Btuh): <sup>31</sup>								
3.14 Door orientation (N, NE, E, SE, S, SW, W, NW): <sup>32</sup>								
3.15 Sensible Heat Gain At Design Conditions (kBtuh): <sup>31</sup>								
3.16 Latent Heat Gain At Design Conditions (kBtuh):								
3.17 Total Heat Gain at Design Conditions (kBtuh): <sup>31</sup>	<u> </u>							
3.18 Total Heat Loss at Design Conditions (kBtuh):								



### Appendix A – Supplementary tables for Section 3

3.19 Common Space Heating &	& Cooling Loads (required for all common space	e heating and	cooling systems)	
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)
Common Space Name:	Design Conditions: Total Heat Gain:	(kBtuh)	Total Heat Loss:	(kBtuh)

3.20 Building Heating & Cooling Loads (only required when shared systems such as central boilers or chillers are specified)								
System Name:	Design Conditions: Total Heat Gain:(kBtuh)	Total Heat Loss:(kBtuh)						
System Name:	Design Conditions: Total Heat Gain:(kBtuh)	Total Heat Loss:(kBtuh)						
System Name:	Design Conditions: Total Heat Gain:(kBtuh)	Total Heat Loss:(kBtuh)						
System Name:	Design Conditions: Total Heat Gain:(kBtuh)	Total Heat Loss:(kBtuh)						



### Appendix A – Supplementary tables for Section 4

4. Heating & Cooling Equipment Selection									
<b>Cooling Equipment</b> (Complete all a (columns), identical data is not require								e spaces □ N/A	
List Cooling Equipment ID in the spaces to the right; duplicating as needed for each unique space served:									
4.4 Equipment type: (PTAC / AC, Chille WLHP / GSHP / ASHP / VRF)	r / CT, PTHP /								
4.5 Area / Space(s) that system serves:									
4.6 Chiller / condenser / outdoor unit ma	anufacturer:								
4.7 Chiller / condenser / outdoor unit me	odel #:								
4.8 Evaporator / indoor unit manufactur	er:								
4.9 Evaporator / indoor unit model #:									
4.10 AHRI reference #: 41									
4.11 Rated efficiency:									
4.12 Evaporator fan type: PSC, ECM / I									
4.13 Compressor speed: Single, Two, \									
4.14 Turn down ratio (for variable speed									
4.15 Latent capacity at design condition						_			
4.16 Sensible capacity at design condit									
4.17 Total capacity at design conditions	· · ·								
4.18 Cooling sizing % = Total capacity (Item 4.17) divided by Total Heat Gain of space(s) in Item 4.5:									
4.19 Meets cooling sizing limit: (A, B, C, D or N/A) <sup>29, 31</sup>									
4.20 lf "B", list Load sensible heat ratio = Max. sensible heat gain (Item 3.15) / Max. total heat gain (Item 3.17): <sup>39</sup>									
4.21 If "B", calculate HDD / CDD ratio: <sup>4</sup>	13								
				21	(Per Item 4	,			
Equipment Type & Climate Condition	Single-Spe	eed	Two-Speed Variable-Sp						
A: For Cooling-Only Equipment or For Cooling Mode of Heat Pump in Condition A Climate <sup>43</sup>	Recommended Allowed: 90		Recommended: 90 – 120% Allowed: 90 – 140%		Recommended: 90 – 130% Allowed: 90 – 160%				
B: For Cooling Mode of Heat Pump in Condition B Climate <sup>43</sup>	90% - 100%, pl	us 15 kBtuh	90% - 100%, plus 15 kBtuh			90% - 100%, plus 15 kBtuh			
C: For low-load spaces (≤15 kBtuh) <sup>44</sup>	≤ 20 kB	Stuh							
D: For low-load spaces (≤18 kBtuh) <sup>44</sup>				≤ 25 kBtu	h		≤ 25 kBtuh		
Heating Equipment (Complete all a multiple spaces (columns), identical c check "N/A".)								🗆 N/A	
List Heating Equipment ID in the spaces duplicating as needed for each unique s									
4.22 Electric equipment type: PTHP, W VRF, Boiler, Furnace, Electric Res									
4.23 Gas Equipment type: HW PTAC / fan coil, Gas-Fired PTAC, Boiler, Furnace									
4.24 Area / Space(s) that system serves:									
	S:								
4.25 Manufacturer:	S:								
4.25 Manufacturer: 4.26 Model Number:	S:								
4.25 Manufacturer: 4.26 Model Number: 4.27 AHRI reference #: <sup>41</sup>	S:								
<ul> <li>4.25 Manufacturer:</li> <li>4.26 Model Number:</li> <li>4.27 AHRI reference #: <sup>41</sup></li> <li>4.28 Rated efficiency:</li> </ul>									
<ul> <li>4.25 Manufacturer:</li> <li>4.26 Model Number:</li> <li>4.27 AHRI reference #: <sup>41</sup></li> <li>4.28 Rated efficiency:</li> <li>4.29 Equipment output capacity (kBtuh)</li> </ul>	:								
<ul> <li>4.25 Manufacturer:</li> <li>4.26 Model Number:</li> <li>4.27 AHRI reference #: <sup>41</sup></li> <li>4.28 Rated efficiency:</li> </ul>	: city (kBtuh) (17°F):								



4.32 Furnace heating sizing % = Total capacity (Item 4.29) divided by Total Heat Loss of space(s) in Item 4.24:							
4.33 Meets furnace sizing limit: (A, B, C, or N/A) <sup>29</sup>							
A: For low-load spaces	A: For low-load spaces (≤ 10 kBtuh), furnace output capacity is ≤ 40 kBtuh						
B: When Used for Heating Only	C: When Paired With Cooling						
100 – 400%	Recommended: 100 – 140% Allowed: 100 – 400%						

#### Appendix A – Supplementary tables for Section 5

5. Dwelling-Unit Duct Design				
5.2 Room-by-room design airflows doo	cumented below (which mus	sum to the mode with the higher Design HVAC fan airflow). <sup>12, 47, 48</sup>		
Name of the unit plan:		Name of the unit plan:		
Name of the unit plan: Design HVAC fan airflow: <sup>49</sup>		Name of the unit plan:		
	ting mode CEM	Design HVAC fan airflow: <sup>49</sup> Cooling mode CFM Heating mode CFM		
		Cooling mode CFM Heating mode CFM Design HVAC fan speed setting (e.g., low, medium, high): <sup>50</sup>		
Design HVAC fan speed setting (e.g., low, medium, high): 50         Cooling mode       Heating mode		Cooling mode Heating mode		
Design total external static pressure (corresponding to the mode with the higher airflow above): <sup>51</sup> IWC		Design total external static pressure (corresponding to the mode with the higher airflow above): <sup>51</sup> IWC		
Room Name	Design Airflow (CFM)	Room Name Design Airflow	(CFM)	
1		1		
2		2		
3		3		
4		4		
5		5		
6		6		
7		7		
8		8		
9		9		
10		10		
Total for all roor	ns	Total for all rooms		
Name of the unit plan:		Name of the unit plan:		
Design HVAC fan airflow: <sup>49</sup>		Design HVAC fan airflow: 49		
Cooling mode CFM Heating mode CFM		Cooling mode CFM Heating mode CFM		
Design HVAC fan speed setting (e.g., low, medium, high): <sup>50</sup> Cooling mode Heating mode		Design HVAC fan speed setting (e.g., low, medium, high): <sup>50</sup> Cooling mode Heating mode		
Design total external static pressure (corresponding to the mode		Design total external static pressure (corresponding to the mode		
with the higher airflow above): <sup>51</sup> IWC		with the higher airflow above): <sup>51</sup> IWC		
Room Name	Design Airflow (CFM)	Room Name Design Airflow	(CFM)	
1		1		
2		2		
3		3		
4		4		
5		5		
6		6		
7		7		
8		8		
9		9		
10		10		
Total for all roor	ns	Total for all rooms		
	-			