ENERGY STAR[®] Residential New Construction Programs

Historical Document

This document is provided for reference because it has been superseded by a more recent Version or Revision. Please find current program documents on the <u>Program</u> <u>Requirements</u> webpage.

Use of older Versions and Revisions, such as this document, are typically limited to homes and buildings with a permit date (or, for manufactured homes, a production date) prior to a specified date. Consult the <u>Implementation Timeline</u> table to assess whether a home or apartment is still eligible to be certified using this document.

For questions or more information, contact us at <u>energystarhome@energystar.gov</u>.



HVAC Designer Responsibilities:

•	Complete one National HVAC Design Report for each system design for a house plan, created for either the specific plan configuration (i.e.,
	elevation, option, orientation, & county) of the home to be certified or for a plan that is intended to be built with different configurations (i.e.,
	different elevations, options, and/or orientations). Visit <u>www.energystar.gov/newhomeshvacdesign</u> and see Footnote 2 for more information.

• Obtain efficiency features (e.g., window performance, insulation levels, and infiltration rate) from the builder or Rater.³

• Provide the completed National HVAC Design Report to the builder or credentialed HVAC contractor and to the Rater.

1. Design Overview									
1.1 Designer name:									
1.2 Select which party you are providing these desig	n service	s to:	🗆 Build	er	or		Credentiale	ed HVAC co	ontractor
1.3 Name of company you are providing these design services to (if different than Item 1.1):									
1.4 Area that system serves: □ Whole-house	🗆 Up	per-level		ower-level		Other			
1.5 Is cooling system for a temporary occupant load			□ Yes	🗆 No					
1.6 House plan:		Check box	to indicate	e whether t	he system	design is	site-specif	ic or part of	a group: ²
□ Site-specific design. Option(s) & elevation(s) modeled:									
Group design. Group #: out of	total grou	ups for this	s house pla	n. Con	figuration r	nodeled:			
2. Dwelling Unit Mechanical Ventilation System Design ("Vent System") ^{5, 6, 7} & Inlets in Return Duct ⁸									Designer Verified
Airflow:									
2.1 Ventilation airflow design rate & run-tim						later. ⁹			
2.2 Ventilation airflow rate required by 62.2				CFM					-
2.3 Design for this system: Vent. airflow ra	te:	_CFM_R	lun-time pe	er cycle:	minute	es Cycle	time:	minutes	-
System Type & Controls:									
2.4 Specified system type:		xhaust	🗆 Bala	inced					-
2.5 Specified control location:							oath, utility	room)	-
2.6 Specified controls allow the system to operate automatically, without occupant intervention.									
2.7 Specified controls include a readily-accessible ventilation override and a label has also been specified if its function is not obvious (e.g., a label is required for a toggle wall switch, but not for a switch that's on the ventilation equipment).									
2.8 For any outdoor air inlet designed to connect to a ducted return of the HVAC system, specified controls automatically restrict airflow using a motorized damper during ventilation off-cycle and occupant override. ^{8, 10}									
Sound: 2.9 The fan of the specified system is rated	≤ 3 sone	s if interm	ittent and ≤	1 sone if o	continuous	, or exem	oted. 11		
Efficiency:									
2.10 If Vent System controller operates the 4.7 is ECM / ICM or the controls will red									
2.11 If bathroom fans are specified as part of the system, then they are ENERGY STAR certified. ¹³									
Air Inlet Location: (Complete this section if system	has a spe	ecified air i	inlet locatio	on; otherwis	se check "N	J/A"). ¹⁴			D N/A
2.12 Inlet pulls ventilation air directly from c	utdoors a	and not fro	m attic, cra	wlspace, g	garage, or a	adjacent d	welling un	it.	
2.13 Inlet is ≥ 2 ft. above grade or roof deck; ≥ 10 ft. of stretched-string distance from known contamination sources (e.g., stack, vent, exhaust, vehicles) not exiting the roof, and ≥ 3 ft. from known sources exiting the roof.									
3. Room-by-Room Heating & Cooling Loads ¹⁵									
3.1 Room-by-room loads calculated using: Unabri	dged AC	CA Manua	lJv8 □	2013 ASH	IRAE Fund	amentals	□ Other	per AHJ ¹⁶	-
3.2 Indoor design temperatures used in loads are 70	°F for hea	ating and T	75°F for co	oling.					
3.3 Outdoor design temperatures used in loads: (See Footnote 17 and energystar.gov/hvacdesigntemps) ¹⁷								-	
County & State, or US Territory, selected: Cooling season:°F Heating season:°F									
3.4 Number of occupants used in loads: ¹⁸								-	
3.5 Conditioned floor area used in loads: ¹⁹ Sq. Ft.									-
3.6 Window area used in loads: ²⁰ Sq. Ft.									-
3.7 Predominant window SHGC used in loads: ²¹								-	
3.8 Infiltration rate used in loads: ²² Summer: Winter:								-	
3.9 Mechanical ventilation rate used in loads:								-	
Loads At Design Conditions (kBtuh)	Ν	NE	E	SE	S	SW	W	NW	-
3.10 Sensible heat gain (By orientation ²³):									-
3.11 Latent heat gain (Not by orientation):		·	·					-	
Cooling 3.12 Total heat gain (By orientation ²³):									-
3.13 Maximum – minimum total heat gain (Item 3.12) across orientations =kBtuh Variation is ≤ 6 kBtuh. ^{23, 24}									
Heating 3.14 Total heat loss (Not by orientation):								-	



4.1 Equipment selected per ACCA Manual S (see Footnele 26.2 × 20). ^{82.9} □ 4.2 Equipment type: □ Cooling-only air conditioner of neuron will be installed; otherwise check 'NAA') □ NA 4.2 Equipment type: □ Cooling-only air conditioner of neuron will be installed; otherwise check 'NAA') □ 4.3 Condenser manufacturer & model: □ □ 4.5 AHRI todeance & .'' □ □ 4.6 Control State State Personal Control State S	4. Heating & Cooling Equipment Selection ¹⁵									Designer Verified	
4.2 Equipment type:	4.1 Equipment selected per ACC	CA Manual S	(see Fo	ootnote 25 & 26). 25, 2	6						
4.3 Condenser manufacturer & model:	Air Conditioner / Heat Pump	(Complete i	f air cor	nditioner or heat pu	mp will	be installed; otl	herwise	chec	k "N/A")		D N/A
4 E vaporator / fan coll manufacturer & model: -<									-		
4.5 AHRI reference #: ** - 4.6 AHRI reference file COP 4.7 Evaporator fan type: D Single-speed Uvariable-speed - 4.8 Compressor type: D Single-speed Uvariable-speed - 4.9 Latent capacity at design conditions, from OEM expanded performance data: ** KBuh - 4.10 Sensible capacity at design conditions, from OEM expanded performance data: ** KBuh - 4.11 Total capacity at design conditions, from OEM expanded performance data: ** KBuh - 4.12 Contrag straing % = Total capacity (Item 4.11) divided by maximum total heat gain (Item 3.12) % - 4.14 Complete this Item if Condition B Climate will be used to select sizing limit in tem 4.15. Otherwise, check *NA*: ** - - 4.14 Load sensible heat ratio = Max. sensible heat gain (Item 3.10) / Max. total heat gain (Item 3.12) = % - 4.15 Colling Strain design focation Per Item 4.21 Single-Speed Two-Speed Variable-Speed Variable-Speed Compressor Type (Per Item 4.21) Single-Speed Two-Speed Variable-Speed - For Cooling Mode of Heat Pump in Cooling sizing limit (4.15). Imax Cooling sizing % Imax Cooling sizing % - - 4.16 Load sing Sizing									-		
46 AHR listed efficiency: / EER / SEER / Air-source heat pump: HSP Ground-source heat pump: COP - 47 Evaporator fan type: D Sngle-speed Warible-speed Other: - 4.9 Compressor type: D Single-speed Two-speed Variable-speed - 4.9 Latent capacity at design conditions, from OEM expanded performance data: ** KBtuh - 4.11 Total capacity at design conditions, from OEM expanded performance data: ** KBtuh - 4.14 Compressor type: N/A - ** KBtuh - 4.14 Compressor type: N/A - ** KBtuh - 4.14 Congressor type: N/A - ** ** - 4.14 Compressor type: N/A - ** ** - 4.14 Compressor type: Compressor type: ** - - ** 4.14 Compressor type: Compressor type: ** -										-	
4.7 Exponent na type: D PSC D ECM / ICM D Other: - 4.8 Compressor type: D Single-speed Variable-speed - 4.10 Sensible capacity at design conditions, from OEM expanded performance data: * KBtuh - 4.10 Sensible capacity at design conditions, from OEM expanded performance data: * KBtuh - 4.11 Total capacity at design conditions, from OEM expanded performance data: * KBtuh - 4.12 Ansource heat pump capacity K17/F: KBtuh N/A - 4.14 Complete this Item if Condition B Climate will be used to select sizing limit item 4.15. Otherwise, check *NA*: ** - - 4.14 Complete this Item if Condition B Climate will be used to select sizing limit item 4.15. Otherwise, check *NA*: ** - - 6.14.2 Condition Climate Condition Per lem 4.2) & Compressor Type (Per Item 4.8) - - Conding-Only Equipment Type (Per Item 4.2) & Recommended: 90 - 100%, Allowed: 90 - 1	4.5 AHRI reference #: 27			_							-
48 Compressor type: □ Single-speed □ Variable-speed	4.6 AHRI listed efficiency:	_/	EER / S	EER Air-source he	at pum	o: HSPF G	iround-s	source	heat pump:0	COP	-
49 Latent capacity at design conditions, from OEM expanded performance data: ²⁸ kBtuh 4.10 Sensible capacity at design conditions, from OEM expanded performance data: ²⁸ kBtuh 4.11 Total capacity at design conditions, from OEM expanded performance data: ²⁸ kBtuh 4.12 Arr.source heat pump capacity: M177: kBtuh NA 4.12 Arr.source heat pump capacity: M177: kBtuh NA 4.14 Carnsource heat pump capacity: Camabacki (time 4.11) divided by maximum total heat gain (time 3.12): % . 4.14 Carnsource heat pump capacity: Camabacki (time 4.11) divided by maximum total heat gain (time 3.12): % . 4.14 2.100 / CDD ratio (Visit energystar gev/hvacdes/grimms to determine this value for the design location) = . . 4.15 Check bxx of applicable cooling sizing limit from chart below: ^{28, 28} Compressor Type (Per Item 4.3) . For Cooling-ONIP department or For Cooling Mode of Heat Pump in Condition 8 Climate Condition 8 Climate A flowed: 90 - 100%, plus 15 kBtuh Condition 8 Climate . . .<										-	
4 10 Sensible capacity at design conditions, from OEM expanded performance data: ²⁸ kBuh 4.11 Total capacity at design conditions, from OEM expanded performance data: ²⁸ kBuh 4.12 Ar-source heat pump capacity: At 17*F: kBuh NA 4.13 Cooling sizing % = Total capacity (tem 4.11) divided by maximum total heat gain (tem 3.12): % A 4.14 Complete his tem if Condition B Climate will be used to select sizing limit time 41.5. Otherwise, check 'NA': ²⁸ NA 4.14 2 HDD / COD raio (Visit energystar cav/ivacdes/grittemps to determine this value for the design location) = 4.15 Check tox of applicable colling sizing limit from chart below? Recommended: 90 – 120% Allowed: 90 – 140% Variable-Speed For Cooling-Orly Equipment of Per Cooling Mode of Heat Pump in Condition A Climate Recommended: 90 – 120% 										-	
411 Total capacity at design conditions, from OEM expanded performance data: "** kBuh - 4.12 Air-source heat pump capacity. At 17F: kBuh NA 4.13 Cooling sizing % = Total capacity (Item 4.11) divided by maximum total heat gain (Item 3.12). % - 4.14 Complete this Item If Condition B Climate will be used to select sizing limit in Item 4.15. Otherwise, check TNA*: 78 NA 4.14 Complete this Item If Condition B Climate will be used to select sizing limit in Item 4.15. Otherwise, check TNA*: 78 NA 4.14 2 HDD / CDD ratio (Visit energystar.cov/hyacdesigntemps to determine this value for the design location) = - 4.15 Check box of applicable cooling sizing limit from chart below: *5 79 Compressor Type (Per Item 4.3) Climate Condition (Per Item 4.14) Single-Speed Tow-Speed Variable-Speed For Cooling Mode of Heat Pump in Conding sizing limit (for chart below: *5 79 Recommended: 90 - 120% Recommen										-	
4.12 Air-source heat pump capacity: At 17*F: kBtuh At 47*F: kBtuh NA 4.13 Cooling sizing % = Total capacity (Item 4.11) divided by maximum total heat gain (Item 3.12): % - 4.14 Complete this them if Condition B Climate will be used to select sizing limit in them 4.15. Otherwise, check "NIA": NA 4.14 Complete this them if Condition B Climate will be used to select sizing limit in them 4.15. Otherwise, check "NIA": NA 4.14 Complete this them if Condition B Climate will be used to select sizing limit from chart below: ^{39, 29} - 4.15 Check box of applicable cooling sizing limit from chart below: ^{39, 29} Compressor Type (Per Item 4.2) - Climate Condition (Per Item 4.14) Single-Speed Two-Speed Variable-Speed For Cooling Mode of Heat Pump in Condition A Climate Single-Speed Two-Speed Variable-Speed For Cooling Mode of Heat Pump in Condition A Climate 90% - 100%, plus 15 kBtuh 00% - 100%, plu									-		
4.13 Cooling sizing % = Total capacity (Item 4.11) divided by maximum total heat gain (Item 3.12): % 4.14 Complete this Item if Condition B Climate will be used to select sizing limit in Item 4.15. Otherwise, check 'N/A': ²⁸ N/A N/A 4.14.2 HDD / CDD ratio (Visit energystar.gov/hyvaddssigntemps to determine this value for the design location) =	4.11 Total capacity at design co	nditions, from	OEM e	expanded performan	ce data:	28			kBtu	h	-
4.14 Complete this Item if Condition B Climate will be used to select sizing limit in tem 4.15. Otherwise, hock "N/A": ²⁰ N/A 4.14.1 Load sensible heat ratio = Max. sensible heat gain (litem 3.10) / Max. total heat gain (litem 3.12) =	4.12 Air-source heat pump capa	icity: At	17°F:	kBtuh		At 47°F:		kB	tuh 🗆 N	/A	-
4.14.1 Load sensible heat ratio = Max. sensible heat gain (Item 3.10) / Max. total heat gain (Item 3.12) =% - 4.14.2 HDD / CDD ratio (Visit energystar gov/hvaddesigniemps to determine this value for the design location) =	4.13 Cooling sizing % = Total ca	apacity (Item	4.11) di	vided by maximum to	otal hea	t gain (Item 3.12	2):	%			-
4.14.1 Load sensible heat ratio = Max. sensible heat gain (Item 3.10) / Max. total heat gain (Item 3.12) =% - 4.14.2 HDD / CDD ratio (Visit energystar gov/hvaddesigniemps to determine this value for the design location) =	4.14 Complete this Item if Cond	ition B Climat	e will be	e used to select sizin	g limit ir	n Item 4.15. Othe	erwise,	check	"N/A": ²⁹ □ N/	A	
4.14.2 HDD / CDD ratio (Visit energystar gov/hvacdesigntemps to determine this value for the design location) = 4.15 Check box of applicable cooling sizing limit from chart below: ^{25, 24} - Cuimate Condition (Per Item 4.14) Single-Speed Two-Speed Variable-Speed For Cooling Mode of Heat Pump in Condition A Climate Recommended: 90 – 115% Allowed: 90 – 140% Recommended: 90 – 130% Allowed: 90 – 140% 90% - 100%, plus 15 kBtuh - 410 Clicaapacity: - 410 Litaapacity: - <td< td=""><td></td><td></td><td></td><td></td><td></td><td></td><td></td><td></td><td></td><td>,</td><td>-</td></td<>										,	-
Compressor Type (Per Item 4.8)									ation) =		
Compressor Type (Per Item 4.8)			-						,		-
Climate Condition (Per term 4.14) Single-Speed Two-Speed Variable-Speed For Cooling Mode of Heat Pump in Condition A Climate Recommended: 90 – 130% Recommended: 90 – 140% Recommended: 90 – 130% For Cooling Mode of Heat Pump in Condition A Climate 90% - 100%, plus 15 kBtuh 100 10					Con	pressor Type (F	Per Item	4.8)			
For Cooling, Mode of Heat Pump in Condition A Climate Recommended: 90 – 110% Allowed: 90 – 140% Recommended: 90 – 120% Allowed: 90 – 140% Recommended: 90 – 130% Allowed: 90 – 140% For Cooling Mode of Heat Pump in Condition B Climate 90% - 100%, plus 15 kBtuh 90% - 100%, plus 15 kBtuh 90% - 100%, plus 15 kBtuh 4.16 Cooling sizing % (4.13) is within cooling sizing limit (4.15).			Si	nale-Speed				,	Variable-	Spee	d
For Cooling Mode of Heat Pump in Condition A Climate Image: Superstand Supe	•	.,	-	<u> </u>	+	·				<u> </u>	
Condition À Climate Allowed. 90 = 100% Allowed. 90 = 140% Allowed. 90 = 140% For Cooling Mode 9 Heat Pump in Condition B Climate 90% - 100%, plus 15 kBtuh 90% - 100%, plus 15 kBtuh 90% - 100%, plus 15 kBtuh 4.16 Cooling sizing % (4.13) is within cooling sizing limit (4.15). Image: State 100% Image: State 100% Furnace (Complete if furnace will be installed; otherwise check "N/A"). N/A 4.17 Furnace manufacturer & model:		in 🗆							i		
Condition B Climate D 90% - 100%, plus 15 KButh 90%	-		All	owed: 90 – 130%		Allowed: 90	- 140%	σ	Allowed:	90 – 1	160%
Furnace (Complete if furnace will be installed; otherwise check "N/A"). IN/A 4.17 Furnace manufacturer & model: - 4.18 Listed efficiency: - 4.19 Total capacity: - 4.20 Heating sizing % = Total capacity (Item 4.19) divided by total heat loss (Item 3.14): % 4.21 Check box of applicable heating sizing limit from chart below: - When Used for Heating Only When Paired With Cooling 4.22 Heating sizing % (4.20) is within heating sizing limit (4.21). - 5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D. Cooling modeCFM 5.2 Design HVAC fan airflow: - 5.3 Design HVAC fan speed setting (e.g., low, medium, high): 22 Cooling modeCFM 5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): 33 - 7 12 23 - - 6 17 28 - - 5		o in 🛛	90% -	100%, plus 15 kBtul	ח ח	90% - 100%, pl	us 15 k	Btuh	□ 90% - 100%,	plus	15 kBtuh
4.17 Furnace manufacturer & model:											
4.18 Listed efficiency:	Furnace (Complete if furnace	will be instal	led; oth	nerwise check "N/A").						D N/A
4.19 Total capacity: ³⁰	4.17 Furnace manufacturer & m	odel:									-
4.20 Heating sizing % = Total capacity (Item 4.19) divided by total heat loss (Item 3.14):% - 4.21 Check box of applicable heating sizing limit from chart below: - When Used for Heating Only When Paired With Cooling 100 - 140% Recommended: 100 - 140% Allowed: 100 - 400% 4.22 Heating sizing % (4.20) is within heating or cooling equipment will be installed with ducts; otherwise check "N/A"). ¹⁵ N/A 5. Duct Design (Complete if heating or cooling equipment will be installed with ducts; otherwise check "N/A"). ¹⁵ N/A 5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D.	4.18 Listed efficiency:			AF	UE						-
4.21 Check box of applicable heating sizing limit from chart below: - When Used for Heating Only When Paired With Cooling 100 - 140% Recommended: 100 - 140% Allowed: 100 - 400% 4.22 Heating sizing % (4.20) is within heating sizing limit (4.21). Imit (4.21) 5. Duct Design (Complete if heating or cooling equipment will be installed with ducts; otherwise check "N/A"). 15 N/A 5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D. Imit (4.21) 5.2 Design HVAC fan airflow: ³¹ Cooling mode CFM 5.3 Design HVAC fan speed setting (e.g., low, medium, high): ³² Cooling mode Heating mode - 5.4 Design total external static pressure (corresponding to the mode with the higher airflow in Item 5.2): ³³ IWC - 5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): ^{34, 35} - Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) Poend Airflow (CFM) 1 12 23 - - - - 3 14 24 - - - - 4 15 26 - - - - -	4.19 Total capacity: ³⁰			kBtuh							-
When Used for Heating Only When Paired With Cooling 100 - 140% Recommended: 100 - 140% Allowed: 100 - 400% 4.22 Heating sizing % (4.20) is within heating sizing limit (4.21). Image: Cooling equipment will be installed with ducts; otherwise check "N/A"). ¹⁵ N/A 5. Duct Design (Complete if heating or cooling equipment will be installed with ducts; otherwise check "N/A"). ¹⁵ N/A 5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D. Image: CFM Heating mode Image: CFM 5.2 Design HVAC fan airflow: ³¹ Cooling mode Heating mode Image: CFM - 5.4 Design total external static pressure (corresponding to the mode with the higher airflow in Item 5.2): ³³ IWC - 5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): ^{34, 35} - Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) Image: CFM 1 12 23 Image: CFM Image: CFM Image: CFM 3 14 25 Image: CFM Image: CFM Image: CFM 4 15 26 Image: CFM Image: CFM Image: CFM Image: CFM 4 15 26 </td <td>4.20 Heating sizing % = Total ca</td> <td>apacity (Item</td> <td>4.19) di</td> <td>vided by total heat lo</td> <td>ss (Iten</td> <td>ו 3.14):</td> <td>_ %</td> <td></td> <td></td> <td></td> <td>-</td>	4.20 Heating sizing % = Total ca	apacity (Item	4.19) di	vided by total heat lo	ss (Iten	ו 3.14):	_ %				-
Image: 100 - 140% Image: Recommended: 100 - 140% Allowed: 100 - 400% 4.22 Heating sizing % (4.20) is within heating sizing limit (4.21). Image: Recommended: 100 - 140% Allowed: 100 - 400% 5. Duct Design (Complete if heating or cooling equipment will be installed with ducts; otherwise check "N/A"). ¹⁵ N/A 5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D. Image: CFM Image: CFM 5.2 Design HVAC fan airflow: ³¹ Cooling mode CFM Heating mode CFM 5.3 Design HVAC fan speed setting (e.g., low, medium, high): ³² Cooling mode CFM Heating mode - 5.4 Design total external static pressure (corresponding to the mode with the higher airflow in Item 5.2): ³⁴ . ³⁵ - - 5.5 Room-by-room design airflow documented below (which must sum to the mode with the higher airflow in Item 5.2): ³⁴ . ³⁵ - Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) 1 12 23 - - 2 13 24 - - 3 14 25 - - 4 15 26 - - <t< td=""><td>4.21 Check box of applicable he</td><td>ating sizing li</td><td>mit fron</td><td>n chart below:</td><td></td><td></td><td></td><td></td><td></td><td></td><td>-</td></t<>	4.21 Check box of applicable he	ating sizing li	mit fron	n chart below:							-
4.22 Heating sizing % (4.20) is within heating sizing limit (4.21). □ 5. Duct Design (Complete if heating or cooling equipment will be installed with ducts; otherwise check "N/A"). ¹⁵ □ N/A 5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D. □ 5.2 Design HVAC fan airflow: ³¹ Cooling modeCFM Heating modeCFM - 5.4 Design total external static pressure (corresponding to the mode with the higher airflow in Item 5.2): ³³ IWC - - 5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): ^{34, 35} - - Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) 1 12 23 - - - - 2 13 24 - - - 3 14 25 - - - 4 15 26 - - - 5 16 27 - - - - 6 17 28 - - - - - 9 20 31 <td>When Used for</td> <td>Heating Only</td> <td></td> <td></td> <td></td> <td>When</td> <td>Paired</td> <td>With (</td> <td>Cooling</td> <td></td> <td></td>	When Used for	Heating Only				When	Paired	With (Cooling		
5. Duct Design (Complete if heating or cooling equipment will be installed with ducts; otherwise check "N/A"). 15 IN/A 5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D. Image: CFM Image: CFM 5.2 Design HVAC fan airflow: ³¹ Cooling mode CFM Heating mode CFM Image: CFM 5.3 Design HVAC fan speed setting (e.g., low, medium, high): ³² Cooling mode CFM Heating mode CFM Image: CFM 5.4 Design total external static pressure (corresponding to the mode with the higher airflow in Item 5.2): ³³ IWC Image: CFM Image: CFM Image: CFM 5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): ^{34, 35} Image: CFM	□ 100 -	– 140%				Recommended:	100 – 14	40%	Allowed: 100 – 40	0%	
5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D.											
5.1 Duct system designed for the equipment selected in Section 4, per ACCA Manual D.									D N/A		
5.3 Design HVAC fan speed setting (e.g., low, medium, high): ³² Cooling mode Heating mode - 5.4 Design total external static pressure (corresponding to the mode with the higher airflow in Item 5.2): ³³ IWC - - 5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): ^{34, 35} - - Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) 1 12 23 - - - - 2 13 24 - - - 3 14 25 - - - 4 15 26 - - - 5 16 27 - - - 6 17 28 - - - 7 18 29 - - - 8 19 30 - - - 9 20 31 - - -		-	-						,		
5.3 Design HVAC fan speed setting (e.g., low, medium, high): ³² Cooling mode Heating mode I - 5.4 Design total external static pressure (corresponding to the mode with the higher airflow in Item 5.2): ³³ IWC - - 5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): ^{34, 35} - - Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) 1 12 23 - - - - 2 13 24 - - - - 3 14 25 - <td>5.2 Design HVAC fan airflow: ³¹</td> <td></td> <td></td> <td>Coo</td> <td>ling mo</td> <td>de CF</td> <td>-M F</td> <td>leatin</td> <td>g mode 🛛 🔾</td> <td>CFM</td> <td>-</td>	5.2 Design HVAC fan airflow: ³¹			Coo	ling mo	de CF	-M F	leatin	g mode 🛛 🔾	CFM	-
5.4 Design total external static pressure (corresponding to the mode with the higher airflow in Item 5.2): ³³ IWC - 5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): ^{34, 35} - Room Name Design Airflow (CFM) Room Name Design Airflow (CFM)	-	ting (e.g., low	, mediu								-
5.5 Room-by-room design airflows documented below (which must sum to the mode with the higher airflow in Item 5.2): ^{34, 35} - Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) Room Name Design Airflow (CFM) 1 12 23 - 2 13 24 - 3 14 25 - 4 15 26 - 5 16 27 - 6 17 28 - 7 18 29 - 8 19 30 - 9 20 31 - 10 21 32 -									•		-
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Footnotes

This report is designed to meet ASHRAE 62.2-2010 or later and ANSI / ACCA's 5 QI-2015 protocol, thereby improving the performance of HVAC equipment in new homes when compared to homes built to minimum code. However, these features alone cannot prevent all ventilation, indoor air quality, and HVAC problems (e.g., those caused by a lack of maintenance or occupant behavior). Therefore, system designs documented through the use of this report are not a guarantee of proper ventilation, indoor air quality, or HVAC performance.

This report applies to split air conditioners, unitary air conditioners, air-source heat pumps, and water-source (i.e., geothermal) heat pumps up to 65 kBtuh with forced-air distribution systems (i.e., ducts) and to furnaces up to 225 kBtuh with forced-air distribution systems (i.e., ducts). For all other permutations of equipment (e.g., boilers, mini-split / multi-split systems) and distribution systems, Section 1 and 2 are required and Sections 3 through 5 are recommended, but not required.

- 2. The report shall represent a single system design for a house plan. Check the box for "site-specific design" if the design was created for the specific plan configuration (i.e., elevation, option, orientation, and county) of the home to be certified. Check the box for "group design" if the design was created for a plan that is intended to be built with potentially different configurations (i.e., different elevations, options, and/or orientations). Regardless of the box checked, the system design as documented on this National HVAC Design Report must fall within the following tolerances for the home to be certified:
 - Item 3.3: The outdoor design temperature used in loads are within the limits defined at energystar.gov/hvacdesigntemps.
 - Item 3.4: The number of occupants used in loads is within ± 2 of the home to be certified.
 - Item 3.5: The conditioned floor area used in loads is between 100 sq. ft. smaller and 300 sq. ft. larger than the home to be certified.
 - Item 3.6: The window area used in loads is between 15 sq. ft. smaller and 60 sq. ft. larger than the home to be certified, or, for homes to be certified with >500 sq. ft. of window area, between 3% smaller and 12% larger.
 - Item 3.7: The predominant window SHGC is within 0.1 of the predominant value in the home to be certified.
 - Items 3.10 3.12: The sensible, latent, & total heat gain are documented for the orientation of the home to be certified.
 - Item 3.13: The variation in total heat gain across orientations is ≤ 6 kBtuh.
 - Item 4.16: The cooling sizing % is within the cooling sizing limit selected.

Provide the National HVAC Design Report to the party you are providing these design services to (i.e., a builder or credentialed HVAC contractor) and to the Rater. The report is only required to be provided once per system design, even if multiple homes are built using this design (e.g., in a production environment where the same plan is built multiple times, only one report is required). As long as a report has been provided that falls within these tolerances for the home to be certified, no additional work is required. However, if no report falls within these tolerances or if any aspect of the system design changes, then an additional report will need to be generated prior to certification.

Visit <u>energystar.gov/newhomeshvacdesign</u> for a tool to assist with group designs and for more information.

- 3. The term 'Rater' refers to the person(s) completing the third-party verification required for certification. The person(s) shall: a) be a Certified Rater or Approved Inspector, as defined by ANSI / RESNET / ICC 301, or an equivalent designation as determined by a Home Certification Organization (HCO); and, b) have attended and successfully completed an EPA-recognized training class. See www.energystar.gov/newhomestraining.
- 4. Check "Yes" if this system is to handle temporary occupant loads. Such a system may be required to accommodate a significant number of guests on a regular or sporadic basis and shall be handled by a supplemental cooling system (e.g., a small, single-package unit or split-coil unit) or by a system that can shift capacity from zone to zone (e.g., a variable volume system).
- 5. As defined by ANSI / RESNET / ICC 301-2019, a Dwelling Unit Mechanical Ventilation System is a ventilation system consisting of powered ventilation equipment such as motor-driven fans and blowers and related mechanical components such as ducts, inlets, dampers, filters and associated control devices that provides dwelling-unit ventilation at a known or measured airflow rate.
- 6. The system shall have at least one supply or exhaust fan with associated ducts and controls. Local exhaust fans are allowed to be part of a Dwelling Unit Mechanical Ventilation System. Designers may provide supplemental documentation as needed to document the system design.
- 7. In "Warm-Humid" climates as defined by 2009 IECC Figure 301.1 (i.e., CZ 1 and portions of CZ 2 and 3A below the white line), it is recommended, but not required, that equipment be specified with sufficient latent capacity to maintain indoor relative humidity at ≤ 60%.
- 8. Item 2.8 applies to any outdoor air inlet connected to a ducted return of the dwelling unit HVAC system, regardless of its intended purpose (e.g., for ventilation air, make-up air, combustion air). This Item does not apply to HVAC systems without a ducted return. For example, if an outdoor air inlet connected to a ducted return is used as a dedicated source of outdoor air for an exhaust ventilation system (e.g., bath fan), the outdoor airflow must be automatically restricted when the exhaust fan is not running and in the event of an override of the exhaust ventilation system. Note that a Rater will generally measure the ventilation rate at the highest HVAC fan speed applicable to ventilation mode (e.g., if the inlet only opens when the HVAC is in 'fan-only' mode, it will be tested in this mode) to verify that it is ≤ 15 CFM or 15% above design value. As an alternative, measurement of the outdoor airflow can be waived if a Constant Airflow Regulating (CAR) damper with a manufacturer-specified maximum flow rate no higher than 15 CFM or 15% above the ventilation design value is installed on the inlet.
- 9. Airflow design rates and run-times shall be determined using ASHRAE 62.2-2010 or later. Designers are permitted, but not required, to use published addenda and/or more recent editions of the standard to assess compliance.
- 10. In addition, consult manufacturer requirements to ensure return air temperature requirements are met.
- 11. Dwelling Unit Mechanical Ventilation System fans shall be rated for sound at no less than the airflow rate in Item 2.3. Fans exempted from this requirement include HVAC air handler fans, remote-mounted fans, and intermittent fans rated ≥ 400 CFM. To be considered for this exemption, a remote-mounted fan must be mounted outside the habitable spaces, bathrooms, toilets, and hallways and there shall be ≥ 4 ft. ductwork between the fan and intake grill. Per ASHRAE 62.2-2010, habitable spaces are intended for continual human occupancy; such space generally includes areas used for living, sleeping, dining, and cooking but does not generally include bathrooms, toilets, hallways, storage areas, closets, or utility rooms.
- 12. Note that the 'fan-on' setting of a thermostat would not be an acceptable controller because it would continuously operate the HVAC fan.
- 13. Bathroom fans with a rated flow rate \geq 500 CFM are exempted from the requirement to be ENERGY STAR certified.



- 14. Without proper maintenance, ventilation air inlet screens often become filled with debris. Therefore, EPA recommends, but does not require, that these ventilation air inlets be located so as to facilitate access and regular service by the occupant.
- 15. Homes certified through the Caribbean Program Requirements, Version 3, are exempt from completing Sections 3, 4, and 5 of this report.
- 16. Select "2013 ASHRAE Fundamentals" if using Chapter 17 of the 2013 ASHRAE Handbook of Fundamentals. Select "Other per AHJ" if the Authority Having Jurisdiction where the home will be certified mandates the use of a load calculation methodology other than Unabridged ACCA Manual J v8 or 2013 ASHRAE Fundamentals.
- 17. Visit <u>energystar.gov/hvacdesigntemps</u> for the maximum cooling season design temperature and minimum heating season design temperature permitted for ENERGY STAR Single-Family New Homes. For "County & State, or US Territory, selected", select the County and State or US Territory (i.e., Guam, Northern Mariana Islands, Puerto Rico, or US Virgin Islands), where the home is to be certified. The same design report is permitted to be used in other counties, as long as the design temperature limits in those other counties meet or exceed the cooling and heating season temperature limits for the county selected. For example, if Frederick County, VA, is used for the load calculations, with a 1% cooling temperature limit of 93 °F, then the same report could be used in Fairfax County (which has a higher limit of 94 °F) but not in Albemarle County (which has a lower limit of 92 °F). If a jurisdiction-specified design temperature is used that exceeds the limit in the Design Temperature Limit Reference Guide, designers must submit a Design Temperature Exception Request available at <u>energystar.gov/hvacdesigntemps</u>.
- 18. To determine the number of occupants among all HVAC systems in the home, calculate the number of bedrooms, as defined below, and add one. This number of occupants must be within ± 2 of the home to be certified, unless Item 1.5 indicates that the system is a cooling system for temporary occupant loads.

A bedroom is defined by ANSI / RESNET / ICC 301-2014 as a room or space 70 sq. ft. or greater size, with egress window and closet, used or intended to be used for sleeping. A "den", "library", or "home office" with a closet, egress window, and 70 sq. ft. or greater size or other similar rooms shall count as a bedroom, but living rooms and foyers shall not.

An egress window, as defined in 2009 IRC section R310, shall refer to any operable window that provides for a means of escape and access for rescue in the event of an emergency. The egress window definition has been summarized for convenience. The egress window shall:

- have a sill height of not more than 44 inches above the floor; AND
- have a minimum net clear opening of 5.7 sq. ft., height of 24 in., and width of 20 in.; AND
- be operational from the inside of the room without the use of keys, tools or special knowledge.
- 19. The difference between the Conditioned Floor Area (CFA) used in the design and the actual home to be certified must fall within the tolerance specified in Footnote 2, as verified by a Rater. Be advised, the Rater will calculate CFA using the definition in ANSI / RESNET / ICC 301-2019, which defines this value, in part, as the floor area of the Conditioned Space Volume within a building or Dwelling Unit, not including the floor area of attics, crawlspaces, and basements below air sealed and insulated floors. See https://codes.iccsafe.org/content/chapter/16185/ for the complete definition.
- 20. The difference between the window area used in the design and the actual home to be certified must fall within the tolerance specified in Footnote 2, as verified by a Rater. Be advised, the Rater will calculate window area using the on-site inspection protocol provided in Normative Appendix B of ANSI / RESNET / ICC 301-2019, which instructs the Rater to measure the width and height of the rough opening for the window and round to the nearest inch, and then to use these measurements to calculate window area, rounding to the nearest tenth of a square foot. See https://codes.iccsafe.org/content/chapter/16191/ for the complete protocol.
- 21. "Predominant" is defined as the SHGC value used in the greatest amount of window area in the home.
- 22. Infiltration rate shall reflect the value used in the confirmed or projected ERI rating for home to be certified. Alternatively, use "Average" or "Semiloose" values for the cooling season infiltration rate and "Semi-tight" or "Average" values for the heating season infiltration rate, as defined by ACCA Manual J, Eighth Edition, Version Two.
- 23. Orientation represents the direction that the front door of the house is facing. The designer is only required to document the loads for the orientation(s) that the house might be built in. For example, if a house plan will only be built one time in a specific orientation (e.g., a site-specific design), then the designer only needs to document the loads for this one orientation.
- 24. Determine the orientation with the largest and smallest Total Heat Gain. Verify that the difference in Total Heat Gain between the orientation with the largest and smallest value is ≤ 6 kBtuh. If not, then assign the orientations into one or more groups until the difference is ≤ 6 kBtuh and then complete a separate National HVAC Design Report for each group.
- 25. Equipment shall be selected using the maximum total heat gain in Item 3.12 and the total heat loss in Item 3.14 per ACCA Manual S, Second Edition, except that cooling ranges above ACCA Manual S limits are temporarily allowed, per Item 4.15.
- 26. As an alternative for low-load spaces, a system match-up including a single-speed compressor with a total capacity ≤ 20 kBtuh is permitted to be used in spaces with a total cooling load ≤ 15 kBtuh. A system match-up including a two-speed or variable-speed compressor with a total capacity ≤ 25 kBtuh is permitted to be used in spaces with a total cooling load ≤ 18 kBtuh.
- 27. If an AHRI Reference # is not available, OEM-provided documentation shall be attached with the rated efficiency of the specific combination of indoor & outdoor components of the air conditioner or heat pump, along with confirmation that the components are designed to be used together.
- 28. The full system capacity at design conditions, from OEM expanded performance data, shall be listed. For two-speed equipment, the full system capacity shall reflect the capacity at the maximum available compressor speed. For variable-speed equipment, it shall reflect the capacity when the compressor operates at the AHRI rating speed.
- 29. Per ACCA Manual S, Second Edition, if the load sensible heat ratio is ≥ 95% and the HDD/CDD ratio is ≥ 2.0, then the Climate is Condition B, otherwise it is Condition A.
- 30. The full system capacity shall be listed. For two-stage and modulating furnaces, the full system capacity shall reflect the maximum output available.
- 31. Design HVAC fan airflow is the design airflow for the blower in CFM, as determined using the manufacturer's expanded performance data.
- 32. Design HVAC fan speed setting is the setting on the control board (e.g., low, medium, high) corresponding to the Design HVAC fan airflow.



- 33. Design total external static pressure is the pressure corresponding to the Design HVAC fan airflow, inclusive of external components (e.g., evaporator coil, whole-house humidifier, or ≥ MERV 6 filter).
- 34. Designers may provide supplemental documentation with room-by-room and total design airflows in lieu of completing Item 5.5. Sample supplemental documentation can be found at http://www.energystar.gov/newhomeshvacdesign.
- 35. Orientation-specific room-by-room design airflows are recommended, but not required, to distribute airflow proportional to load, thereby improving comfort and efficiency.

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